

Lab 2 (associated with Hw 2): Motor spin-down test

The objective of this laboratory is to gain physical insights into 1st-order, linear, ODEs and to recognize how motor angular speed ω is influenced by:

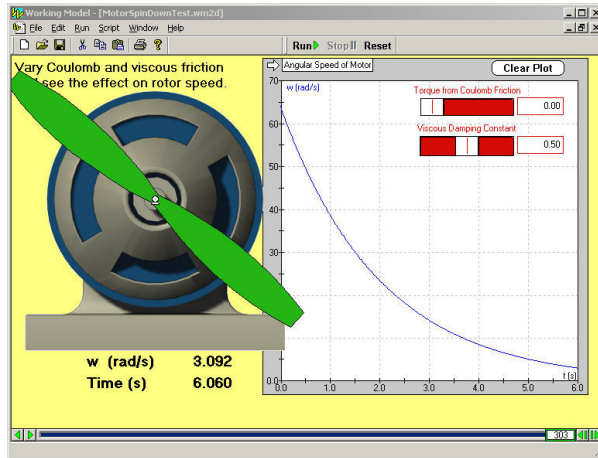
- b , the motor's linear viscous damping constant
- T_f , the Coulomb friction torque on the motor

and to determine numerical values for b and T_f from experimental data (*system identification*).

Lab 2.1 Effect of viscous friction and Coulomb friction on a motor's speed.

To begin this problem, double-click on the Working Model file MotorSpinDownTest.wm2d.

In the following table, record τ_c and t_{stop} or $t_{settling}$ in units of seconds.



To answer each question with Working Model, click the **Reset** button (if necessary) and click and drag the sliders that control the values of b (measured in n*m*sec) and T_f (measured in n*m).

To start the simulation, click the **Run** button, and to stop it, click the **Stop** button.

For finer control of simulation time, use the **◀** arrow or **▶** arrows to rewind or advance the simulation.

For each simulation that follows, use the plot of the motor's angular speed versus time to determine:

- How $\omega(t)$ decreases (circle linear, exponential, both, or neither)
- τ_c , the time required for the motor's speed to decrease to $e^{-1} \approx 0.37$ of its initial speed
- t_{stop} , the time it takes for the motor to stop spinning
- $t_{settling}$, the time required for $\omega(t)$ to settle within 1% of ω_{ss} [the steady-state value of $\omega(t)$], i.e., $t_{settling}$ is the minimum value of t such that for $t \geq t_{settling}$, $|\omega(t) - \omega_{ss}| \leq 0.01 * |\omega_{ss} - \omega(0)|$.

Note: Since $\omega_{ss} = 0$ and $\omega(0) = 64$, $t_{settling}$ is the time required for $\omega(t) \leq 0.64$.

If $t_{stop} \neq \infty$, skip $t_{settling}$.

| | $T_f=0$ | $T_f=15$ | $T_f=30$ |
|---------|--|--|--|
| $b=0$ | linear/exponential $\tau_c = \infty$ $t_{stop} = \infty$ | linear/exponential $\tau_c =$ <input type="text"/> $t_{stop} =$ <input type="text"/> | linear/exponential $\tau_c =$ <input type="text"/> $t_{stop} =$ <input type="text"/> |
| $b=0.5$ | linear/exponential $\tau_c =$ <input type="text"/> $t_{settling} =$ <input type="text"/> | linear/exponential $\tau_c = 1.13$ $t_{stop} = 2.28$ | linear/exponential $\tau_c =$ <input type="text"/> $t_{stop} =$ <input type="text"/> |
| $b=1.0$ | linear/exponential $\tau_c =$ <input type="text"/> $t_{settling} =$ <input type="text"/> | linear/exponential $\tau_c =$ <input type="text"/> $t_{stop} =$ <input type="text"/> | linear/exponential $\tau_c = 0.56$ $t_{stop} = 1.14$ |

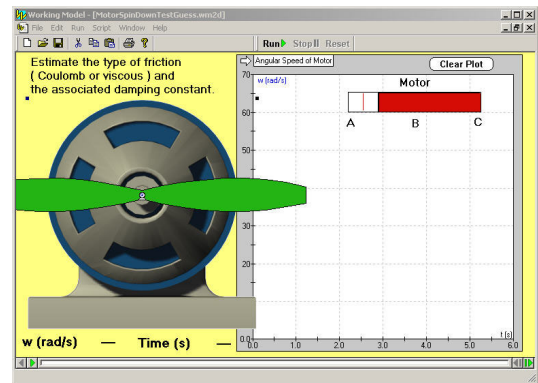
Based on your observations, circle the correct answer in the following statements:

- Linear viscous damping causes the motor's speed to decrease **linearly/exponentially**
- Coulomb friction causes the motor's speed to decrease **linearly/exponentially**
- For a motor that has both linear viscous damping and Coulomb friction, the dominant reason the motor slows down at *small* values of ω is probably **viscous damping/Coulomb friction**, whereas at *large* values of ω , it is probably **viscous damping/Coulomb friction**
- When the motor drives a *propeller*, a better model of the total torque $T_{resistance}$ that slows the motor is $T_{resistance} = T_f + b\omega + c\omega^2$ where c is a constant. The physical phenomenon responsible for the $c\omega^2$ term in driving a propeller is mostly likely .
- At very large values of ω , the dominant term in $T_{resistance}$ is $T_f/b\omega/c\omega^2$ (circle one).

Lab 2.2 Determining the linear viscous damping constant and Coulomb friction torque.

One task performed by an engineer when building or using a motor is determining numerical values for b and T_f from laboratory data and from a known value of I , the moment of inertia of the motor and its attachments about its axis.

For each motor in MotorSpinDownTestGuess.wm2d, determine:



- How $\omega(t)$ decreases (circle linear, exponential, or both)
- What causes the decrease in $\omega(t)$ (circle viscous, Coulomb, or both)
- Numerical values for τ_c , and either t_{stop} or $t_{settling}$ (in seconds).
- Based on your values of τ_c , t_{stop} , and $t_{settling}$, find exact values for b (n*m*sec) and T_f (n*m) for motors A and B (use $I = 1 \text{ kg}\cdot\text{m}^2$).
- The point of this lab is to be able to determine numerical values for b and T_f from experimental data (*system identification*). You have run several simulations to correlate the effect of b and T_f on system response. In addition, you determined b and T_f for Motor A and Motor B. However, neither Motor A or B has both viscous and Coulomb friction. You now do a system identification on a more realistic motor having both viscous and Coulomb friction.

Use your values of τ_c , t_{stop} , and $t_{settling}$, find approximate values for b and T_f for motor C.

Note: Homework 2.13 analyzes the problem of finding b and T_f for motor C.

| Motor A | Motor B | Motor C |
|---|---|---|
| linear/exponential | linear/exponential | linear/exponential |
| viscous/Coulomb | viscous/Coulomb | viscous/Coulomb |
| $\tau_c = $ | $\tau_c = $ | $\tau_c = $ |
| $t_{stop} = $ | $t_{stop} = $ | $t_{stop} = $ |
| $t_{settling} = $ | $t_{settling} = $ | $t_{settling} = $ |
| $b = $ | $b = $ | $b \approx $ |
| $T_f = $ | $T_f = $ | $T_f \approx $ |